AMENDMENTS TO THE SPECIFICATION:

Please replace paragraph [0016] with the following amended paragraph:

[0016] The support arm 18 is provided with a general U-shape and includes a forward portion 75, a rearward portion 77, a body section 78 and a pair of downwardly-extending arms 80 opposed across body section 78. Each arm **[[**78**]]** 80 is provided with a first aperture 82 located proximate forward portion 75, a second aperture 84 located proximate rearward portion 77, and an arcuately-shaped slot 86 located proximate rearward portion 77 and below aperture 84.

Please replace paragraph [0017] with the following amended paragraph

[0017] In assembly, a pivot shaft 88 is received within the apertures 32 of the mounting bracket 24 and the apertures 84 of the support arm **[[**88**]]** 18, thereby pivotally connecting the rearward portion 77 of the support arm 18 to the mounting bracket 24 at a point 90 as defined by a longitudinal axis of the pivot shaft 88. A pivot shaft 92 extends through the apertures 30 of the mounting bracket 24 and the slots 86 of the support arm 18, thereby operably coupling the rearward portion 77 of the support arm 18 to the mounting bracket 24 at a point 94 as defined by a longitudinal axis of the pivot shaft 92. A pivot shaft 96 extends through the apertures 82 of the support arm 18 and the apertures 64 of the first section 40 of the platform assembly 16, thereby pivotally coupling the rearward portion 44 of the first section 40 with the forward portion 75 of the support arm 18 at a point 98 as defined by a longitudinal axis of the pivot shaft 96. The

pivot shaft 54 is received within the apertures 60 of the first section 40 of the platform assembly 16 and the apertures 73 of the second section 46, thereby pivotally coupling the rearward portion 50 of the second section 46 with the forward portion 42 of the first section 40 at the point 52. A pivot pin 146 is received within the apertures 62 of the first section 40 of the platform assembly 16 and the aperture 144 of the wedge arm 134, thereby pivotally coupling the second end 142 of the wedge arm 134 and the rearward portion 44 of the first section 40 at a point 148 as defined by a longitudinal axis of the pivot pin 146.

Please replace paragraph [0019] with the following amended paragraph:

[0019] The locking mechanism 20 further includes a wedge member 126 having an aperture 128 extending therethrough and slidably receiving the shaft 92 therethrough. The wedge member 126 also includes an angularly-disposed wedge surface **[[**133**]]** 130 and an abutment surface 132 located opposite the wedge surface 130. The locking mechanism 20 further includes a wedge arm 134 having a first end 136 having an elongated slot 138 that slidably receives the shaft 92 therein and an angularly disposed wedge surface 140, and a second end 142 having an aperture 144 extending therethrough and receiving the pivot pin 146 therein. A coil spring 145 extends between the pins 92, 146 keeping the wedge arm 134 in a locked or engaged position as discussed below. It should be noted that this configuration results in a greater

frictional force being developed between the washers 112, the locking plates 116, and the center arm 110 as the load being exerted on the platform assembly 16 increases.

Please replace paragraph [0020] with the following amended paragraph:

[0020] In operation and **[[is]]** as best illustrated in Fig. 3, an upwardly-directed force 150 exerted on the platform assembly 16 causes the platform assembly 16 to pivot about the shaft 96 with respect to the support arm 18 between an in-use position A and an adjustment or release position B. The pivoting movement of the platform assembly 16 causes the shaft **[[62]]** 146 to slide rearwardly from shaft 92, thereby increasing the distance therebetween. The increase in distance between the shaft **[[62]]** 146 and the shaft 92 causes the first end 136 of the wedge arm 134 to slide forward with respect to the shaft 92, thereby causing the wedge surface 140 of the wedge arm 134 to disengage and reduce the force being exerted on the corresponding and mating wedge surface 130 of the wedge member 126. This reduction in force being exerted on the wedge member 126 allows the wedge member 126 to slide along the shaft 92 in a direction away from the washers 112 and locking plates 126, thereby reducing the force being exerted between the abutment surface 132 and the outermost washer 152 and reducing the frictional engagement between the stacked washers 112, the locking plates 116 and the center arm 100. The reduction **[[and]]** of the frictional engagement between the washers 112, the locking plates 116 and the center arm 100 allows the center arm **[[110]]** 100 to slide with respect to the shaft 92, whereas the center arm 100

is normally held in position by the frictional engagement between the locking surfaces 110 of the center arm 100 and the plurality of washers 112 and locking plates 116 when in a normal at rest position and the force 150 is not being applied. The height of the platform assembly 116 with respect to the support surface 14 may then be easily adjusted to a determined height, where the force 150 is removed and the spring 145 biases the shaft 146 towards the shaft 92, thereby forcing the wedge arm 134 to slide rearwardly, the wedge surface 140 of the wedge arm 134 to engage the wedge surface 130 of the wedge member 126, the abutment surface 132 of the wedge member 126 to apply a force to a plurality of washers 112 and the locking plates 116, which in turn frictionally abut the locking surfaces 110 of the center arm 100 and frictionally lock the same from sliding movement with respect to the shaft 92. It should be noted that while the illustrated example includes a plurality of eight washers 112, and a plurality of six locking plates, the magnitude of the frictional force exerted across the plurality of the washers 112 and the locking plates 116 and ultimately exerted on the center arm 110 may be increased by increasing the number of the washers 112 and the locking plates 116. It should be noted that the biasing force exerted by the spring 145 on the shaft 146 prevents accidental loss of support of the platform assembly 16, as may be caused by an operator exerting a downwardly-directed force on the platform assembly 16 forward of the pivot point 98.